

EE160 2025

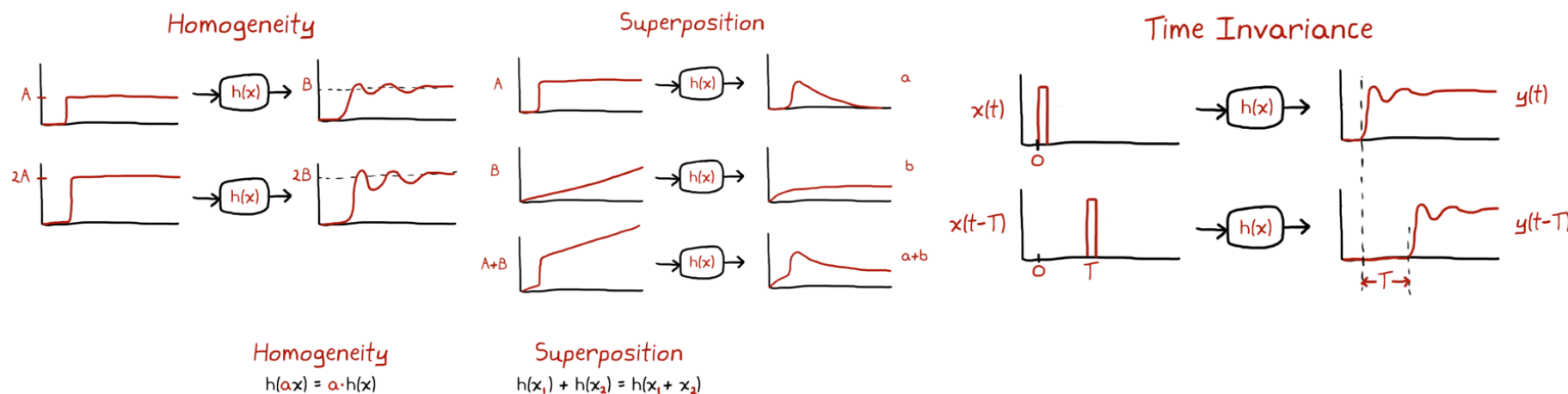
A Quick Review

1线性系统：定义，满足的两个条件

- 叠加原理
- 齐次性

Linear System

- Linearity is defined in terms of the input $u(t)$ and output $y(t)$
 - Also known as excitation and response.
- If a system satisfies **homogeneity** and **superposition**, it is a linear system.
- If a linear system further sanctifies the property of **time-invariance**, it is then called an LTI system.
 - Linearity + Time-invariance = **sinusoidal fidelity**
 - Linear system is a wider concept than *linear time-invariant (LTI)* system, also including *linear time-varying* system



2稳态响应：理解稳态误差对应的表

Steady State Error Specifications

- System type: the number of integrators in the forward path
 - Three error constants are defined for each system type.

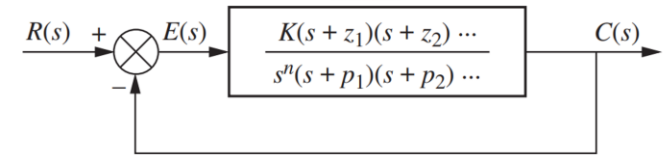


FIGURE 7.8 Feedback control system for defining system type

TABLE 7.2 Relationships between input, system type, static error constants, and steady-state errors

Input	Steady-state error formula	Type 0 $\frac{K(s+z_1)(s+z_2)\dots}{s^0(s+p_1)(s+p_2)\dots}$		Type 1 $\frac{K(s+z_1)(s+z_2)\dots}{s^1(s+p_1)(s+p_2)\dots}$		Type 2 $\frac{K(s+z_1)(s+z_2)\dots}{s^2(s+p_1)(s+p_2)\dots}$	
		Static error constant	Error	Static error constant	Error	Static error constant	Error
Step, $u(t)$	$\frac{1}{1 + \lim_{s \rightarrow 0} G(s)} = \frac{1}{1 + K_p}$	$K_p = \text{Constant}$	$\frac{1}{1 + K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
Ramp, $tu(t)$	$\frac{1}{\lim_{s \rightarrow 0} sG(s)} = \frac{1}{K_v}$	$K_v = 0$	∞	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{\lim_{s \rightarrow 0} s^2G(s)} = \frac{1}{K_a}$	$K_a = 0$	∞	$K_a = 0$	∞	$K_a = \text{Constant}$	$\frac{1}{K_a}$

Interpreting the Steady-State Error Specification

PROBLEM: What information is contained in the specification $K_p = 1000$?

SOLUTION: The system is stable. The system is Type 0, since only a Type 0 system has a finite K_p . Type 1 and Type 2 systems have $K_p = \infty$. The input test signal is a step, since K_p is specified. Finally, the error per unit step is

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + 1000} = \frac{1}{1001} \quad (7.54)$$

3状态空间：可控性、可观性的含义，部分可控系统的极点配置



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Partially Controllable System

□ Further consider this state space model and do what we have done again.

□ Write out the differential equation for x_3 and x_4 :

- $\dot{x}_3 = -2x_3 + 1u$
- $\dot{x}_4 = -2x_4 + 2u$
- Define a new state denoted by $\Delta x = 2x_3 - x_4$,
- which is governed by $\frac{d}{dt}\Delta x = -4x_3 + 2x_4 = -2\Delta x$
- The eigenvalue of the new state Δx cannot be modified by u .
 - This is again what essentially the concept of uncontrollable state.

□ For the first block, let's also cancel the input u by defining a new delta state.

- $\frac{d}{dt}\Delta x = \frac{d}{dt}(x_1 - x_2) = 3x_1 + x_2 - 3x_2 = -3\Delta x + x_2$
- Here, x_2 is affected by u and can be treated as a virtual input.

$$\dot{x} = \begin{bmatrix} 3 & 1 & 0 & 0 \\ 0 & 3 & 0 & 0 \\ 0 & 0 & -2 & 0 \\ 0 & 0 & 0 & -2 \end{bmatrix} x + \begin{bmatrix} 1 \\ 1 \\ 1 \\ 2 \end{bmatrix} u$$

Controllability



A key question that arises in the design of state variable compensators is whether or not all the poles of the closed-loop system can be arbitrarily placed in the complex plane.

The concepts of controllability and observability were introduced by Kalman in the 1960s:

if the system is controllable and observable, then we can.

A system is completely controllable if there exists an unconstrained control $u(t)$ that can transfer any initial state $x(t_0)$ to any other desired location $x(t)$ in a finite time, $t_0 \leq t \leq T$.

For the SISO LTI system

$$\dot{x}(t) = Ax(t) + Bu(t),$$

we can determine whether the system is controllable by examining the algebraic condition

$$\text{rank}[B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B] = n.$$

The controllability matrix P_c is described in terms of A and B as

$$P_c = [B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B],$$

If the determinant of P_c is nonzero, the system is controllable.

dimension of the system.

4 频率响应：波特图、相位裕度与非最小相位系统

- 相位裕度的正负对于非最小相位系统来说是没意义的

Stability Margin in terms of F.R.

- When we are discussing stability, we are implying the stability of the closed loop control system.
- The frequency response of a closed loop transfer function can be derived as follows.

$$T(s) = \frac{G(s)}{1 + G(s)H(s)}$$

$$\Rightarrow T(j\omega) = \frac{G(j\omega)}{1 + G(j\omega)H(j\omega)}$$

- Note this $T(j\omega)$ is the complex gain to a sinusoidal input $R(j\omega)$ and **apparently this gain should not be equal to infinity (as a sufficient condition)**, that is, the open loop transfer function should satisfy:

$$\Rightarrow L(j\omega) \triangleq G(j\omega)H(j\omega) \neq -1 = 1 \angle 180^\circ$$

Gain margin, G_M . The gain margin is the change in open-loop gain, expressed in decibels (dB), required at 180° of phase shift to make the closed-loop system unstable.

Phase margin, Φ_M . The phase margin is the change in open-loop phase shift required at unity gain to make the closed-loop system unstable.

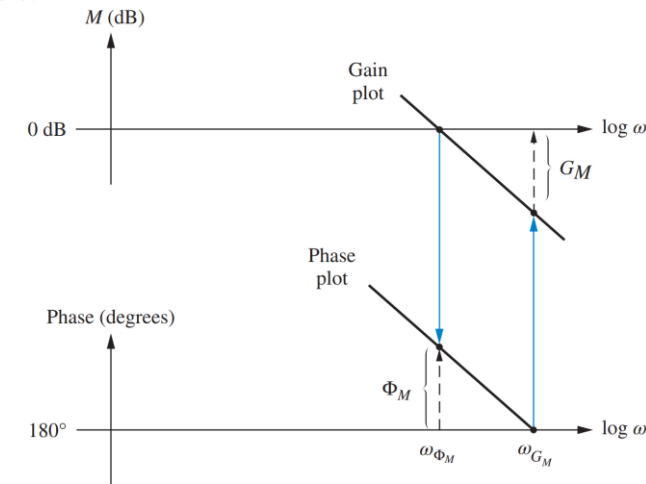


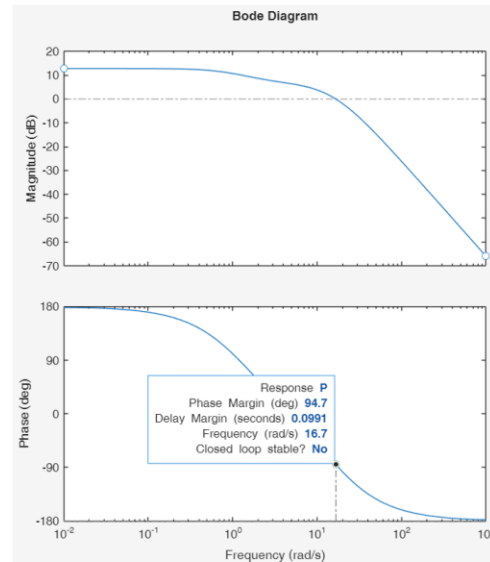
FIGURE 10.37 Gain and phase margins on the Bode diagrams

Nonminimum Phase System due to RHP Zero/Pole

- As a motivation, consider a transfer function, and if its open loop pole or zero is mirrored about the $j\omega$ -axis, i.e., it is moved to the right hand s -plane, the new transfer function shares the same **magnitude** frequency response but its **phase** frequency response would become more lagging or advancing over frequency variable ω .

- An example is provided as script below. Can you check its stability and if it is unstable can you make it stable?

```
close all; cla; clc
s = zpk(0, [], 1);
Popen = 500 * (s+2)/(s+1)/(s^2+30*s+229)
Popen = 500 * (s-2)/(s+1)/(s^2+30*s+229)
P = Popen
allmargin(Popen)
h = bodeplot(Popen);
h.showCharacteristic('AllStabilityMargins')
step(Popen)
P = Popen/(1+Popen); step(P)
```



5. 数学建模：传递函数、状态空间、框图，三者之间相互转换

Block diagram reduction

Feedback

$$G_e(s) = \frac{G(s)}{1 \pm G(s)H(s)}$$

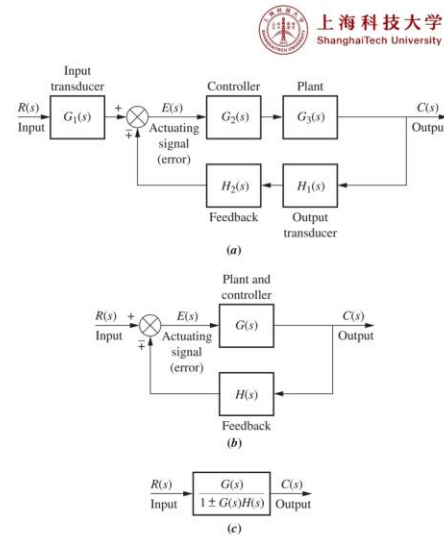


FIGURE 5.6 a. Feedback control system; b. simplified model; c. equivalent transfer function

Transfer Function

General case of a high order differential equation:

$$a_n \frac{d^n c(t)}{dt^n} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \dots + a_0 c(t) = b_m \frac{d^m r(t)}{dt^m} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \dots + b_0 r(t) \quad (2.50)$$

Convert it into s-domain:

$$\begin{aligned} a_n s^n C(s) + a_{n-1} s^{n-1} C(s) + \dots + a_0 C(s) + \text{initial condition terms involving } c(t) \\ = b_m s^m R(s) + b_{m-1} s^{m-1} R(s) + \dots + b_0 R(s) + \text{initial condition terms involving } r(t) \end{aligned}$$

Assume zero initial conditions we get transfer function

$$\frac{C(s)}{R(s)} = G(s) = \frac{(b_m s^m + b_{m-1} s^{m-1} + \dots + b_0)}{(a_n s^n + a_{n-1} s^{n-1} + \dots + a_0)}$$

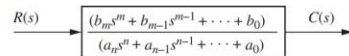


FIGURE 2.2 Block diagram of a transfer function

From transfer function to state space model (1)

Converting a Transfer Function with a Constant Term in the Numerator

$$\frac{C(s)}{R(s)} = \frac{24}{s^3 + 9s^2 + 26s + 24}$$

$$(s^3 + 9s^2 + 26s + 24)C(s) = 24R(s)$$

$$\ddot{c} + 9\dot{c} + 26c = 24r$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -24 & -26 & -9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 24 \end{bmatrix} r$$

$$y = [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

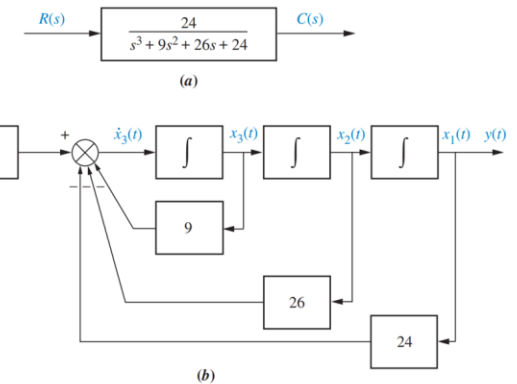


FIGURE 3.10 a. Transfer function; b. equivalent block diagram showing phase variables. Note: $y(t) = c(t)$.

6. 稳定性： 劳斯判据

Routh-Hurwitz Criterion

- Simply stated, the **Routh-Hurwitz criterion** declares that the number of roots of the polynomial that are in the right half-plane is equal to the number of sign changes in the first column of Routh table

PROBLEM: Make a Routh table and tell how many roots of the following polynomial are in the right half-plane and in the left half-plane.

$$P(s) = 3s^7 + 9s^6 + 6s^5 + 4s^4 + 7s^3 + 8s^2 + 2s + 6$$

ANSWER:



Routh-Hurwitz Table Generator

Polynomial

Enter polynomial coefficients separated by commas

6,2,8,7,4,6,9,3

$$3s^7 + 9s^6 + 6s^5 + 4s^4 + 7s^3 + 8s^2 + 2s + 6$$

Go

Routh-Hurwitz Table

s^7	3	6	7	2
s^6	9	4	8	6
s^5	$\frac{14}{3}$	$\frac{13}{3}$	0	0
s^4	$-\frac{61}{14}$	8	6	0
s^3	$\frac{787}{61}$	$\frac{392}{61}$	0	0
s^2	$\frac{8004}{787}$	6	0	0
s	$-\frac{1581}{1334}$	0	0	0
1	6	0	0	0

<https://routhhurwitz.streamlit.app/>

7. 瞬态响应：校正器设计（根轨迹法）

Ideal Derivative Compensation (PD)

PROBLEM: Given the system of Figure 9.17, design an ideal derivative compensator to yield a 16% overshoot, with a threefold reduction in settling time.

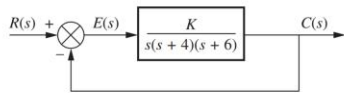


FIGURE 9.17 Feedback control system for Example 9.3

SOLUTION: Let us first evaluate the performance of the uncompensated system operating with 16% overshoot. The root locus for the uncompensated system is shown in Figure 9.18. Since 16% overshoot is equivalent to $\zeta = 0.504$, we search along that damping ratio line for an odd multiple of 180° and find that the dominant, second-order pair of poles is at $-1.205 \pm j2.064$. Thus, the settling time of the uncompensated system is

$$T_s = \frac{4}{\zeta \omega_n} = \frac{4}{1.205} = 3.320 \quad \omega_n = \sqrt{1.204^2 + 2.064^2} = 2.3895$$

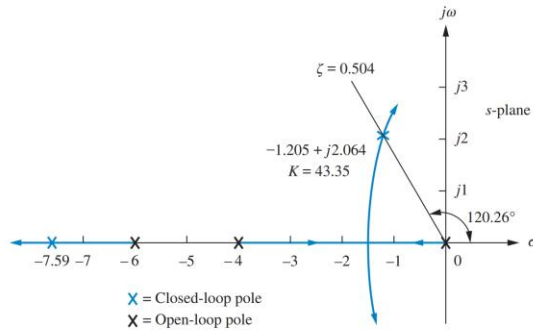


FIGURE 9.18 Root locus for uncompensated system shown in Figure 9.17

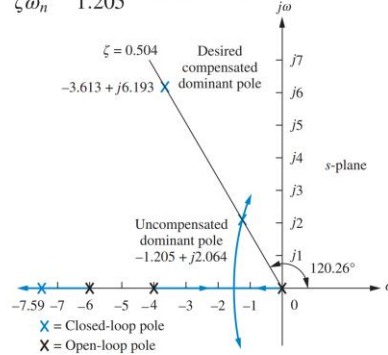


FIGURE 9.19 Compensated dominant pole superimposed over the uncompensated root locus for Example 9.3

猜测 ω_n ，也就是主导极点距离原点的距离，

未动态校正的根这样算： $s = \omega_n(\zeta + j\sqrt{1-\zeta^2})$

特征方程： $1 + G(s) = 0$

$$\Rightarrow G(s) = \frac{K}{s(s+4)(s+6)} = -1$$

验证 s 是否符合角度条件：

$$\Rightarrow \angle G(s) = -\angle s(s+4)(s+6) = 180^\circ$$

确定 s 的值，然后计算希望动态校正到哪里：

$\square =$ 横坐标 + j 纵坐标

- 横坐标基于调节时间 T_s 给定，然后根据 ζ 斜线求纵坐标

- 纵坐标基于峰值时间 T_p 给定，然后根据 ζ 斜线求横坐标

特征方程： $1 + L(s) = 0$

$$\Rightarrow L(s) = C(s)G(s) = \frac{(s+z_c)}{(s+p_c)} \frac{K}{s(s+4)(s+6)} = -1$$

根据角度条件求 PD 控制器的零点放哪里：

$$\Rightarrow \angle L(\square) = \angle(\square + z_c) - (\square + p_c) - \angle \square(\square + 4)(\square + 6) = 180^\circ$$

$$\Rightarrow \angle(\square + z_c) - (\square + p_c) = 180^\circ + \angle \square(\square + 4)(\square + 6)$$

Lag-Lead Compensation

- The velocity error constant of $G_{LC}(s) = \frac{1977}{s(s+10)(s+29.1)}$ is $1977 / 10/29.1 = 6.793$
- The velocity error constant of the uncompensated $G(s) = \frac{192.1}{s(s+6)(s+10)}$ is $\frac{192.1}{6 \times 10} = 3.201$
- Now to improve velocity error constant up to tenfold, we first **arbitrarily** choose the lag compensator pole at 0.01 and then places the lag compensator zero at 0.04713, i.e., $G_{lag}(s) = \frac{(s+0.04713)}{(s+0.01)}$
 - which means the velocity error constant will be improved by $10 / (\frac{6.793}{3.201}) = 4.713$
- The lag-lead compensated open loop system is:

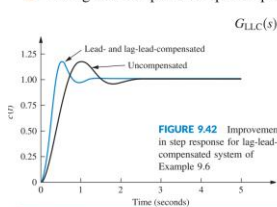


FIGURE 9.42 Improvement in step response for lag-lead-compensated system of Example 9.6

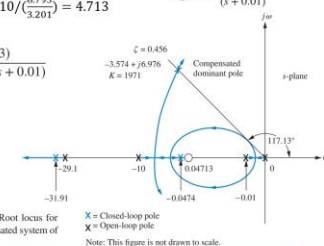


FIGURE 9.41 Root locus for lag-lead-compensated system of Example 9.6

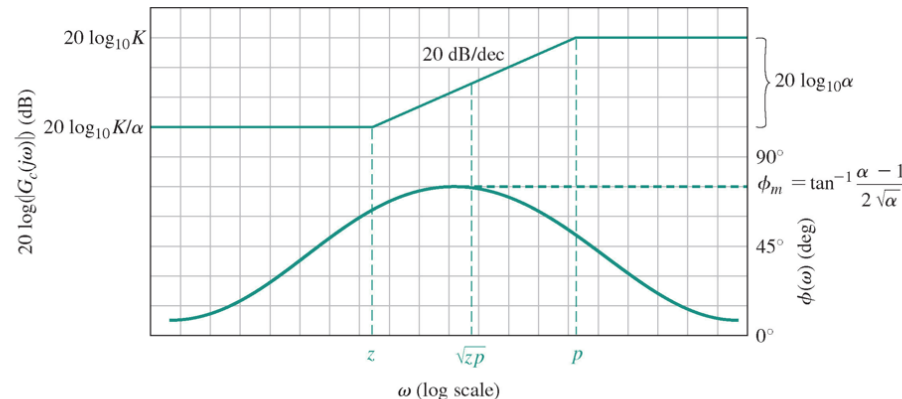
Note: This figure is not drawn to scale.

和频域波特图法)

My Personally Recommended Standard Procedure

- ❑ Consider a unity negative feedback system with a plant

$$P(s) = \frac{1}{s} \frac{1}{0.1s+1} \frac{1}{0.05s+1}$$
 - Try to design cascade compensator such that the **velocity constant** is larger than 20, the **corner frequency** is at least 30 rad/s, and the **phase margin** is at least 50° .
 - Hint: if you find one lead compensator is not enough, consider adding one more.
- ❑ The lag compensator for steady state error is designed after the phase margin requirement has been met.



```

close all;clc;clear all;
s = tf('s');
P = 1 / (s) / (0.05*s+1) / (0.1*s+1);
wc = 30;
K = 10^(-10/20) / abs(evalfr(P, wc*1j));
figure();
bode(K*P)
wm = wc;
alpha = 10;
z1 = sqrt(wm^2/alpha);
p1 = alpha * z1;
C = (s/z1+1) / (s/p1+1);
figure();
bode(C)
OL = C*K*P;
figure();
bode(OL)
allmargin(OL)
    
```

The maximum value of the phase lead occurs at:

$$\omega_m = \sqrt{zp} = \frac{1}{\tau\sqrt{\alpha}}$$